

The turnbuckle shown in Fig. 8-17 has a screw thread with a mean diameter of 12 mm and a lead of 2 mm. If the coefficient of static friction between the screw and the turnbuckle is $\mu_s = 0.25$, determine the moment M that must be applied to draw the end screws closer together.



Fig. 8-17

**SOLUTION**

The moment can be obtained by applying Eq. 8-3. Since friction is not ignored, we must be concerned with Eq. 8-1

$$M = 2rW \tan \phi = dW \quad (1)$$

Here $W = 2000 \text{ N}$, $d = \tan^{-1} \mu_s = \tan^{-1}(0.25) = 14.04^\circ$, $r = 6 \text{ mm}$, and $\theta = \tan^{-1}(2\pi) = \tan^{-1}(12.566) = 85.7^\circ$. Substituting these values into Eq. 1 and solving gives

$$\begin{aligned} M &= 2(2000 \text{ N})(6 \text{ mm}) \tan 14.04^\circ = 3.647 \text{ N}\cdot\text{m} \\ &= 3.65 \text{ N}\cdot\text{m} = 8.33 \text{ N}\cdot\text{m} \end{aligned} \quad (2)$$

NOTE: When the moment is removed, the turnbuckle will be self-locking, i.e., it will not unscrew since $\phi_s > \theta$.

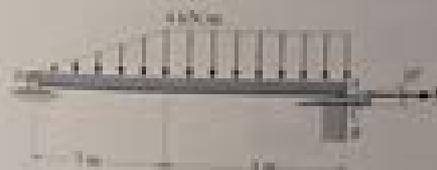
PROBLEMS

8-38. Determine the largest angle θ that will cause the wedge to be self-locking regardless of the magnitude of horizontal force P applied to the blocks. The coefficient of static friction between the wedge and the blocks is $\mu_s = 0.3$. Neglect the weight of the wedge.



Prob. 8-38

8-39. If the beam AB is loaded as shown, determine the horizontal force P which must be applied to the wedge in order to remove it from under the beam. The coefficient of static friction at the wedge's top and bottom surfaces are $\mu_{s1} = 0.25$ and $\mu_{s2} = 0.35$, respectively. If $P = 0$, is the wedge self-locking? Neglect the weight and size of the wedge and the thickness of the beam.



Prob. 8-39

8-40. The wedge has a negligible weight and a coefficient of static friction $\mu_s = 0.15$ with all contacting surfaces. Determine the largest angle θ so that it is self-locking. The response may differ for any magnitude of the force P applied to the pins.



Prob. 8-40

8-41. If the spring is compressed 30 mm and the coefficient of static friction between the tapered wedge C and the slider is $\mu_s = 0.3$, determine the horizontal force P needed to move the slider forward. The wedge is free to move within the slot within the limit angle θ . The coefficient of static friction between A and surface B is $\mu_s = 0.3$. Neglect the weights of the slider and wedge.



Prob. 8-41

8-42. If $P = 20 \text{ N}$, determine the required minimum compression in the spring so that the wedge will not move to the right. Neglect the weight of A and B . The coefficient of static friction for all contacting surfaces is $\mu_s = 0.35$. Neglect friction at the rollers.

8-43. Determine the minimum applied force P required to move wedge A to the right. The spring is compressed a distance of 175 mm. Neglect the weight of A and B . The coefficient of static friction for all contacting surfaces is $\mu_s = 0.35$. Neglect friction at the rollers.



Prob. 8-43

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