M/C DESIGN

- **Length of open belt drive (O.B.D**
$$= 2C + \frac{\pi}{2}(D_1 + D_2) + \frac{(D_2 - D_1)^2}{4C}$$

- **Length of cross belt driven (C.B.D)**
$$= 2C + \frac{\pi}{2}(D_1 + D_2) + \frac{(D_2 + D_1)^2}{4C}$$

- **Velocity ratio**
$$V.R = \frac{N_2}{N_1} = \left(\frac{D_1 + t}{D_2 + t}\right)\left(1 - \frac{S}{100}\right)$$

- **Ratio of belt tension** $\boxed{\dfrac{T_1}{T_2} = e^{\mu\theta}}$

- **Maximum tensile strength of belt**
$$T_{max} = b.t.\sigma_{per}$$

- **Power transmission capacity of belt drive**
$$P.T.C = (T_1 - T_2)v$$

- **Centrifugal tension** $\boxed{T_c = \dfrac{T_{max}}{3}}$

- **Initial tension** $\boxed{T_0 = \dfrac{T_1 + T_2 + 2T_c}{2}}$

- **Number of 'V' belts** $\boxed{n = \dfrac{P_{total}}{P_{each}}} \times K_a$

- **Buckingham dynamic load**
$$F_d = F_t + \frac{20.67v[bc + F_t]}{20.67v + \sqrt{bc + F_t}} \qquad c = \frac{e}{K\left[\dfrac{1}{E_1} + \dfrac{1}{E_2}\right]}$$

- **Soderberg equation** $\boxed{\dfrac{1}{FOS} = \dfrac{\sigma_m}{\sigma_{yt}} + \dfrac{K_f \sigma_v}{\sigma_e}}$

- **Goodman equation** $\boxed{\dfrac{1}{FOS} = \dfrac{K_t\sigma_m}{\sigma_{ut}} + \dfrac{K_f\sigma_v}{\sigma_e}}$

- **Notch sensitivity index** $\boxed{q = \dfrac{K_f - 1}{K_t - 1}}$

- **Unwin's formula** $\boxed{d = 6\sqrt{t}}$

- **Area of transverse fillet weld**
$$\boxed{A_{TFW} = h\ell_e}, \quad \boxed{h = \frac{t}{\cos\theta + \sin\theta}}$$

- $\boxed{\text{Strength of T.F.W} = 0.832 t\,\ell_e\,\tau_t}$

- **Strength of parallel fillet weld**
$$= 0.707t\,\ell_e\,\tau_{per}$$

- **Strength of Butt weld** $= h\ell(\sigma_t)_{per}$

- **Frictional torque of thrust bearing by uniform pressure theory**
$$(T_f)_{UPT} = \frac{2}{3}\mu\omega\left(\frac{R_0^2 - R_i^2}{R_0^2 - R_i^2}\right)$$

- **Frictional torque of thrust bearing by uniform wear theory**
$$(T_f)_{UWT} = \mu\omega\left(\frac{R_0 + R_i}{2}\right)$$

- **Power less** $\boxed{P_{loss} = T_f\,\omega}$

- **T_f equations for flat pivot bearing**
$$(T_f)_{UPT} = \frac{2}{3}\mu WR \qquad (T_f)_{UWT} = \frac{1}{2}\mu WR$$

- **Bearing pressure** $\boxed{P_{ind} = \dfrac{W}{L.D}}$

- **Strength of bearing** $= p_{per} \times L \times D$

- **Bearing characteristic number** $= \dfrac{zn}{p}$

- **Mc-Kee's equation**
$$\mu = \frac{33}{10^8}\left[\left(\frac{zn'}{p'}\right)\left(\frac{D}{C}\right)\right] + K$$

- **Somer field No.** $S = \left(\dfrac{zn}{P}\right)\left(\dfrac{D}{C}\right)^2$

- **Petroff's equation for μ** $\mu = 2\pi^2\left(\dfrac{zn}{P}\right)\left(\dfrac{D}{C}\right)$

- **Life of bearing** $\boxed{\dfrac{L}{90} = \left(\dfrac{C}{P_e}\right)^k}$ (Million rev.)

$K_a = $ service or overload factor.

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Minjie Lin

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Table of Contents Mechanical Engineering All Formulas Machine Design

1. Understanding the eBook Mechanical Engineering All Formulas Machine Design
   - The Rise of Digital Reading Mechanical Engineering All Formulas Machine Design
   - Advantages of eBooks Over Traditional Books
2. Identifying Mechanical Engineering All Formulas Machine Design
   - Exploring Different Genres
   - Considering Fiction vs. Non-Fiction
   - Determining Your Reading Goals
3. Choosing the Right eBook Platform
   - Popular eBook Platforms
   - Features to Look for in an Mechanical Engineering All Formulas Machine Design
   - User-Friendly Interface
4. Exploring eBook Recommendations from Mechanical Engineering All Formulas Machine Design

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